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The mechanical and thermal setup of the GLORIA spectrometer

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C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures



[Back](#)

Close

Full Screen / Esc

[Printer-friendly Version](#)

Interactive Discussion



Abstract

The novel airborne Gimballed Limb Observer for Radiance Imaging of the Atmosphere (GLORIA) measures infrared emission of atmospheric trace constituents. GLORIA comprises a cooled imaging Fourier transform spectrometer which is operated in unpressurized aircraft compartments at ambient temperature. The whole spectrometer is pointed by the gimbal towards the atmospheric target. In order to reach the required sensitivity for atmospheric emission measurements the spectrometer optics needs to operate at a temperature below 220 K. A lightweight and compact design is mandatory due to limited space and high agility requirements. The cooled optical system needs to withstand high pressure and temperature gradients, humidity, and vibrations. A new cooling system based on carbon dioxide and liquid nitrogen combined with high-performance insulation has been developed to meet the mechanical, thermal, and logistical demands. The challenging mechanical and spatial requirements lead to the development of a novel rigid linear slide design in order to achieve the large optical path difference for high spectral resolution. This paper describes the mechanical and thermal setup of GLORIA and presents the performance results on two different research aircrafts.

1 Introduction

The Gimballed Limb Observer for Radiance Imaging of the Atmosphere (GLORIA; Friedl-Vallon et al., 2014) is a cooled Fourier transform infrared (FTIR) spectrometer which has been designed to fly on board the German research aircraft HALO (Krautstrunk and Giez, 2012) or the Russian M55 Geophysica (MDB, 1996). It measures the infrared emission of atmospheric species in the spectral range from 780 to 1400 cm⁻¹. For this purpose the scene is observed through a Michelson linear-slide interferometer and imaged on a detector focal plane array (FPA).

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures



Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



Low temperature operation, as already proved in the former balloon and aircraft borne spectrometers SIRIS (Brasunas et al., 1988), MIPAS-B2 (Friedl-Vallon et al., 2004), MIPAS-STR (Piesch et al., 1996) and CHRISTA-NF (Kullmann et al., 2004), enables the detection of the characteristic infrared emission spectral features of atmospheric species at good signal to noise.

The spectrometer is mounted in a three axes gimbal to provide pointing stability and agility to enable different atmospheric observation modes (Friedl-Vallon et al., 2014): in chemistry mode (CM) with high spectral resolution, the line of sight is nearly perpendicular to the flight direction, whereas in dynamics mode (DM) with high spatial resolution, the instrument scans the horizon stepwise from 45 to 132° in gimbal yaw angle, performing tomographic measurements. The agility of the gimbal also allows nadir measurements and deep space calibration as well as pointing towards the black-body calibration system (Olschewski et al., 2013).

The instrument is operated in unpressurized compartments and observes the atmospheric radiation through an opening on the side of the instrument bay. An installation in the pressurized cabin would require a window outside the calibration path of the instrument. In consequence, the instrument is exposed to the air flow and to variable ambient conditions depending on the flight profile, the aircraft velocity, and the meteorological situation. Overall instrument operating reliability and performance are influenced by environmental parameters such as temperature, humidity, vibration and pressure. A high-speed data acquisition system collects the scientific data as well as a large number of housekeeping data including the ambient conditions and the status of the GLORIA instrument.

The gimballed movement of the instrument requires a lightweight and compact mechanical design of the spectrometer which incorporates an integrated cooling system. In addition, a mechanically rigid structure for the sensitive optics of the interferometer is necessary in order to gain robustness against vibrations which are stimulated aerodynamically at the opening of the instrument bay and by the aircraft itself. The

cooling system has to meet the needs of campaign operation with long flights and short stopovers at airports with limited infrastructure.

Section 2 describes the mechanical and thermal requirements of the airborne GLO-RIA spectrometer. The selected concept and its implementation are presented. The efforts which were made to achieve a stiff and thermally insulated optic module are emphasized. Furthermore the dedicated cooling system and its operation are shown. In Sect. 3 follows a description of the environmental conditions during flights on both the M55 Geophysica aircraft and the G550 HALO aircraft based on sensor data. Finally the performance of the mechanical and thermal system is discussed in Sect. 4.

2 Mechanical and thermal design

2.1 Requirements

The mechanical construction of the spectrometer houses the optical components comprising a 9 cm double sided optical path difference Michelson Interferometer and an infrared detector unit which accepts an incoming beam with a nominal diameter of 36.1 mm and a maximal divergence of 4.1° by 4.1° . Out of the possible configurations for Fourier transform spectrometers (Carli et al., 1999), a single linear-slide two port configuration with cube corners was selected for compactness, robustness, and optical (vignetting) considerations. This optical setup is not shear compensated and needs alignment. Since the instrument is operated in various orientations relative to the direction of gravity it has to be sufficiently stiff enough to preserve this static shear alignment during turning. High stiffness is also needed to suppress vibrational shear that otherwise could be the cause of ghost contributions to the spectra. The target for the velocity stability of the optical path difference measured by the reference laser system is 5 % RMS in order to achieve the required spectroscopic accuracy (Kimmig, 2001). This requirement is based on prior experience with MIPAS-STR.

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



Discussion Paper	Discussion Paper	Discussion Paper	Discussion Paper
------------------	------------------	------------------	------------------

5

10

15

20

25

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

[Back](#)

Close

Full Screen / Esc

[Printer-friendly Version](#)

Interactive Discussion



The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



Discussion Paper	Discussion Paper	Discussion Paper	Discussion Paper
------------------	------------------	------------------	------------------

5

10

15

2.3.1 Thermal insulation

20

25

7, 10965–11010, 2014

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

The image shows a presentation navigation interface with a dark blue background and white text. At the top is a title bar with the text "Title Page". Below it is a grid of eight navigation buttons arranged in four rows and two columns. The first row contains "Abstract" and "Introduction". The second row contains "Conclusions" and "References". The third row contains "Tables" and "Figures". The fourth row contains two buttons with left and right arrow icons. Below the grid is a wide button labeled "Full Screen / Esc". At the bottom are two more wide buttons labeled "Printer-friendly Version" and "Interactive Discussion".

Title Page	
Abstract	Introduction
Conclusions	References
Tables	Figures
◀	▶
◀	▶
Back	Close
Full Screen / Esc	
Printer-friendly Version	
Interactive Discussion	



convection and to protect the cold inner area from the infiltration of humidity and from the formation of condensate.

2.3.2 Insulation feedthrough assembly

The insulation feedthrough assembly, illustrated in Fig. 4 and shown with the cooling tank in Fig. 7, is the interfacing device between the cooled components and the outside environment. It provides a thermally insulated passthrough for all tubes and cables while minimizing heat input from outside to the cooler and optic module.

The insulation feedthrough assembly consists of two plates bonded together with a GFRP spacer tube. The warm end plate is fixed to the pitch level plate and the cold side plate acts as a heat sink cooled by the cold coolant gas exhaust. This design considerably reduces the heat load entering through the outer service connections. The tubes passing through the feedthrough include those for feeding coolant into the cooler system, purging exhaust gas and flushing various interior cavities.

The electrical feedthrough to the optic module consists of two cables and a custom designed combination of multilayer flex cable and printed circuit boards (PCB). In the feedthrough, the cables pass around the cooler exhaust tube and the flex cable sideways down to the pitch level plate. At this location, MICRO-D electrical docking connectors, which are environmentally sealed, provide interface to the outer electronic modules. The intermediary PCB of the flex cable assembly is mounted on the heat sink to further reduce the heat load from outside introduced by the large number of copper wires. The whole feedthrough system is covered by an insulation foam housing and the internal hollow spaces are filled with Styrofoam pellets to minimize convection.

2.3.3 Hygroscopic breathers

The spectrometer's internal spaces have to be pressure-balanced to the environment to avoid forces on the housing and the optic module. Two independent internal hollow spaces are found inside the spectrometer housing: the optic module free space and

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures



Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



the open room between the housing and the optic module, including the interstices between insulation panels. Incoming air has to be dry and clean to protect the optics. Therefore, two hygroscopic breathers filled with molecular sieves and integrated particle filters are connected to the feedthrough assembly. This system allows ventilation, leaving a residual pressure difference of a few hPa while limiting the input of humidity, CO₂ or other contaminants by adsorption. Servicing is done after about five flights by exchanging the filters and by regeneration of the molecular sieves.

2.4 Optic module

The optic module comprises of the optical components of the Michelson interferometer, the imaging lens system, the detector, and the entrance window assembly. The optic module also holds a reference laser system for measuring the optical path difference. The optic module is described in detail below in this section and shown in an exploded view in Fig. 5.

2.4.1 Central body and interferometer optics

The main structure of the optic module, the central body, is milled out of a massive block of fine-grained aluminium which exhibits low thermal distortions related to inner tension. The optic set-up is modular and the subsystems are tightly mounted to the central body. The compact box-like platform makes the interferometer a system which shows high eigen-frequencies and is robust to external vibrations.

The optical components, along the path of the incoming radiation, are the entrance window assembly, the beam splitter unit (BSU), the fixed cube corner, the linear scanner with its moving cube corner, the adjustable infrared objective, and the detector. The supports and fixations for all optical components are made of the same aluminium alloy used for the central body, which leads to uniform thermal contraction during cooling and minimizes misalignments.

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



The infrared radiation enters the optic module via an insulating double window assembly which is hermetically seals the central body. The diameters of the two AR-coated germanium optical windows are 100 mm whereas the detectors full field of view (FOV) at this location forms a rounded square with 78 mm sides. The plates are glued with silicone into two flanges held together by a GFRP tube covered with aluminized Mylar foil to avoid diffusion of water vapour. The outer window can be heated to ensure operation above dew point, especially during ground operation and flight transitions at low altitudes.

The BSU consists of a beam splitter with 104 mm diameter tilted 45° to the incoming beam and a compensation plate with 84 mm diameter. Both the wedged beam splitter substrate and the separated wedged compensation plate (oriented orthogonal to the optical beam) are made of KCl. The plates are held in a rigid aluminium structure by spring retainers.

The fixed cube corner is mounted onto the central body. In order to minimize shear errors, adjustment in the directions perpendicular to the incoming beam is necessary. The position of the fixed cube corner at operating temperature is determined through interferometric measurements. The cube corner positioning then can be reliably adjusted by the use of gauge blocks.

The custom made cube corners are gold coated Zerodur facets fixed by contact bonding (Haisma, Spierings, 2002) with a clear aperture of 72 mm. An adapter is glued to the cube corner. It is made of Invar, which has a much lower thermal expansion coefficient than aluminium, and is therefore compatible with the use of Zerodur.

The reference laser is guided through the interferometer parallel to the infrared beam. It is folded from and back to the reference laser source and detection unit with folding mirrors above and below the imaging optic.

2.4.2 Detector unit

The imaging optic is an air-spaced infrared achromat with a focal length of 72 mm at 218 K. It is mounted in a linear focusing stage in front of the detector. The stage allows

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



movement of 4 mm in axial direction in order to adjust the position of the focal plane to compensate for fabrication tolerances and temperature-dependent focal length variations. The linear focusing stage is driven by a stepper motor with encoder. The torque is transferred by a worm drive and changed to a linear movement by a screw drive. The resulting high gear ratio of 148 turns to 1 mm linear movement of the objective allows fine adjustment and gives enough torque to operate in the specified temperature range.

The detector system, shown in red in Fig. 5, is a custom production by AIM, Heilbronn, Germany. It comprises an FPA detector within a Dewar, the detector front-end electronics and a separate split Stirling cryogenic cooler connected by a helium transfer tube. Together with the imaging optic it forms the detector unit. The high-speed HgCdTe large focal plane array (LFPA) detector with 256 by 256 detector elements is sensitive in the mid and long-wave infrared spectral range between 7 and 12.8 μm . The spectral range is partly limited by the antireflection coated germanium window of the Dewar. The detector is operated at a temperature of 50 K, which is maintained by the 4 W Stirling cooler. All detector unit components are mounted on a holding plate which is fixed to one long-side of the central body. The holding plate and the IR objective belong to the cooled space of the optic module whereas the detector itself and cooler are outside. Therefore, the detector Dewar and its front-end electronic as well as the compressor of the Stirling cooler are thermally insulated from the optic module and rigidly fixed by a GFRP-tube spacer and GFRP supports. Protection against electromagnetic interference is realized by conductive coating of the GFRP-tube, wrapping of the compressor in mu-metal and galvanic isolated mounting.

2.4.3 Scanner

The opto-mechanical setup and especially the scanner has to endure the vibrations under the harsh environmental conditions during flight while holding true to the requirement for velocity variation under 5 % RMS.

The scanner assembly, shown in Fig. 6, was designed to maintain a stiff coupling between the moving cube corner and the central body. A rigid scanner construction was

achieved with a base having high torsion and bending Eigen-frequencies. For the guidance of the slide, a pre-loaded dovetail slide was chosen. Due to its large contact area, it is very stable and largely immune to external loads and vibrations when compared to ball guided slides (Endemann, 1999) or guide slide bearings with small contact areas as used in high resolution laboratory spectrometers (Hase et al., 2013). These advantages are gained on the cost of higher friction; therefore, the material combination and lubrication have to be chosen carefully. For precise guidance of the slide in the dovetail, a lateral pretension is necessary, which is also important to compensate residual thermal expansion. The pretension is achieved by a flexure and supporting compression springs. This simple mechanical solution is flexible in one direction while providing the necessary stiffness along the other directions.

Guidance and carriage are manufactured out of the same aluminium alloy used for all components of the optic module. The guiding surfaces are grinded to get a good surface quality in terms of shape and roughness in order to provide a uniform movement with reduced friction. The surfaces are electroless nickel plated. Pads made out of polytetrafluoroethylene (PTFE) are used on the side of the carriage. The combination of PTFE with nickel leads to low friction with low slip-stick effect. The prefabricated parts were cryogenically cycled before finishing in order to reduce thermal distortion.

For the movement of the slide a thread with 1 mm pitch is used. It provides a stiff coupling between the drive and the slide in the motion direction. The grinded and polished thread is made out of stainless steel. The travelling nut is made out of Polyimide with 15 % molybdenum disulphide (PI MoS₂ 15). The thread is lubricated with lithium based grease, selected for low temperature operation. The thread is supported with a pair of spindle bearings on the drive side and a roller bearing at the opposite end, acting as floating bearing, which are lubricated with perflourinated polyether-based low temperature oil. A lightweight torque servo motor, the Robodrive ILM 50 × 14, is used as a direct drive. Rotor and stator are integrated to the thread and motor housing respectively, leading to a lightweight and compact design. The rotation speed of the thread is measured and controlled by an optical shaft encoder with 32 768 steps per revolution.

10978



The far end points are detected by optical switches. A nominal rotational velocity of 210 encoder steps per millisecond is pre-set which corresponds to a linear slide velocity of 6.41 mm s^{-1} . Due to the factor 2 between the optical and the mechanical path difference, the optical velocity is twice the mechanical velocity.

2.5 Cooling system

In order to fulfil the thermal and logistical requirements described in Sect. 2.1 we decided to cool the optic module by a reservoir filled with dry ice. Charging solid CO_2 into a coolant tank (Piesch et al., 1996; Shallmann et al., 2006; Pint et al., 2001) requires a large opening which disrupts the insulation. Additionally handling, deliverance, and storage of dry ice are costly. We have found a way to avoid these disadvantages by in situ production of dry ice in the coolant tank by expansion of liquid CO_2 (Büst, 2003, E.P. 1429093B1). With this method it is possible to fill liquid CO_2 via a thin non-insulated flexible transfer tube and to store the coolant easily as pressurized CO_2 in gas cylinders. Carbon dioxide supplied in gas cylinders is commercially available. It is ideal for the application during campaigns even in remote areas as it has nearly unlimited storage time and poses little problem for transportation.

Cooling without dry ice filling is also possible by injection of small amounts of liquid CO_2 into the tank. In addition it is possible to use liquid Nitrogen sprayed inside the coolant tank at controlled intervals. Inflight, the cooling system operates with dry ice, whereas liquid N_2 or CO_2 is destined for laboratory and ground operations.

Figure 7 shows the coolant tank construction. A small polyethylene pipe with holes in different zones of the coolant tank is used for filling carbon dioxide. The positions of the holes are optimized to fill the tank volume homogenously with CO_2 snow. A constantan heating wire inside the injection polyethylene pipe protects blocking caused by freezing during the filling procedure. It also allows opening frozen sections of the pipe for the recharging process. A PTFE tube with larger diameter and perforated with holes drains the sublimation gas. The injection pipe and its integrated heating wire are guided and protected inside the PTFE drainage tube.

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



The small holes in the injection pipe act as spray nozzles and in these sections the polyethylene pipe is led outside the PTFE drainage tube. The injection pipe path and size, as well as the number of nozzles, their orientation and their opening have been empirically optimized in several tests to maximize the filled CO₂ snow mass.

The coolant tank has a total volume of 2.2 L and is machined out of a monolithic aluminium block by milling and spark-erosion. The coolant tank is designed for an overpressure of 10 bar to withstand the pressure forces exerted by the dry ice during filling and to operate the coolant tank at pressures up to 5 bar. The overpressure capability allows control of dry ice temperatures from 173 K at 0.1 bar up to 216 K at 5 bar. The construction uses pillars to optimize the pressure resistance. The coolant tank is mounted directly below the optic module, which acts as cover for the tank; this configuration makes optimal use of conductive and convective cooling. The cooler is sealed by phenyl silicon O-rings (Phenyl Vinyl Methyl Quartz or PVMQ).

A separate inlet port on the bottom of the cooling tank enables access with an injection lance of a vacuum insulated transfer hose to spray liquid nitrogen in the tank. For safety reasons two independent pressure relief valves and a bursting disc (not shown on Fig. 7) protect the cooler from overpressure.

2.6 Cooling operation

A functional schematic drawing which shows the GLORIA thermal setup and cooling system with its different cooling possibilities is shown in Fig. 8. During dry ice filling or for controlled cooling, the exhaust port at the pitch level plate hose has to be fully open to release the exhaust gas. After dry ice charging, a smaller hose leading to the aircraft outlet is connected to the main exhaust port.

2.6.1 Controlled cooling

Controlled cooling is achieved by dispersing liquid nitrogen or liquid carbon dioxide inside the coolant tank. The pressure drop and the evaporation of the liquid produce

the desired cooling effect. This procedure can be used to cool down or to maintain the optic module at constant temperature. It is used in the laboratory or during ground operations while on campaign. The liquid nitrogen is taken out of cryogenic storage vessels by a siphon and is transferred to the LN₂ connector with a flexible vacuum insulated transfer line. Alternatively liquid CO₂ is taken out of standard CO₂ cylinders with riser pipe holding liquid CO₂ at room temperature and transferred to the LCO₂ connector. An external feedback loop controller, locked to a temperature sensor on the coolant tank, pilots a magnet valve on the transfer line, injecting an amount of coolant in the tank for controlled time periods. The injected coolant evaporates inside the cooling tank and the exhaust gas is guided out through the PTFE drainage tube.

2.6.2 Dry ice cooling

The optic module is pre-cooled before charging with dry ice. A custom built pressure regulator followed by a non-insulated transfer line provides the liquid CO₂ at 35 bar and slightly below room temperature to the LCO₂ connector. Inside the coolant tank, at the injectors, the CO₂ pressure drops to approximately 1.6 bar resulting in an adiabatic cooling, which leads to the formation of dry ice as well as cold CO₂ gas. Evaporating CO₂ is drained out by the exhaust hose. The produced dry ice snow has a temperature of about 195 K. Approximately 15 L of liquid CO₂ are sufficient to fill to the maximum capacity of 2.3 kg dry ice. Without the use of the injection pipe heater, the pipe and the nozzles freeze and get blocked before the coolant tank is filled at its nominal capacity. The whole filling procedure is monitored by observing temperature, pressure of the system, and the exhaust gas flow. The filling ends by itself when all nozzles are blocked by dry ice, which is clearly identifiable by a sudden drop of the exhaust flow and of the coolant tank internal pressure. At this point the fill valves are closed and the heating of the filling pipe installation is turned off.

Figure 9 shows the measurement of pressure and temperature during dry ice filling and the following warming-up process. In this particular test, the spectrometer was not in operation and thus all internal heat sources were switched off, leaving only heat input

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures



Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



through the housing. The pressure in the coolant tank rises up to 1600 hPa during filling of dry ice. It drops back to nearly ambient pressure after filling and stabilizes to a few hPa above ambient after connecting to the exhaust hose. The ambient temperature is about 294 K. Coolant tank charging needs less than 30 min whereby the temperature drops down to 200 K. The operational phase is characterized by increasing temperatures. This is caused by sublimation of the dry ice which reduces the contact area of the coolant to the tank. Therefore a significant temperature drift of the optic module occurs, even when dry ice remains in the tank. This behaviour is consistent with prior experience with coolers of MIPAS-B2 and MIPAS-STR which use latent heat from phase change of the coolant. The temperature increase rate depends on the coolant tank geometry, the heat load, and the different thermal conductivity of the coolant's phases; the CO₂ gas has a 20 times lower thermal conductivity than dry ice. Figure 9 shows a relative constant temperature drift, rising from 200 to 230 K in 19 h while dry ice remains in the coolant tank. During flight the observed temperature drifts are smaller due to a lower heat load related to the lower environmental temperature (see Sect. 3.3).

Table 2 summarizes the operational characteristics of the GLORIA cooling system for both liquid N₂ and liquid CO₂ cooling, including holding time and consumption. Table 3 shows the occurring heat loads.

3 Results

3.1 Environmental conditions

GLORIA has performed more than 100 flight hours on two different carriers. In this section results from three exemplary flights are presented: Flight 1 on 11 December 2011 on the Geophysica in the polar winter and the flights 8 and 19 on HALO at mid-latitudes on 28 August and on 25 September 2012.

Figure 10 shows temperature and altitude data of one flight with the Geophysica and one flight with HALO. The flight characteristics of both carriers differ in terms of

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



outside pressure with a slight fluctuation correlated with the gimbal yaw angle due to air streaming through the side opening. The differential pressure measurement shows the pressure difference between the inside of the optic module and the belly pod. The pressure fluctuations in the belly pod can also be seen in this differential pressure, causing variations down to -8 hPa. The plot also depicts the pressure variations caused by the breathing of the instrument during ascent, descent, and dives in the range of ± 2 hPa. This is caused by flow resistance through the hygroscopic breathers.

Figure 11c presents the acoustic pressure in the belly pod and the vibrations measured at the pitch level plate. Shown are the RMS values for time periods of 1.2 s during DM measurements and 12 s during CM, respectively. Beside an obvious and expected correlation between the values and the gimbal yaw angle, the mean of these values depend clearly on the flight level. For lower cruise altitudes of 12.6 km at 10:30 UTC the acceleration mean level is about 2.3 m s^{-2} and goes down to 1.1 m s^{-2} at 15 km height at 15:00 UTC. The behaviour of the acoustic pressure is similar. The dependency of the vibrations on the gimbal yaw angle, as well as on the flight altitude, and therefore air density, is explained by aerodynamic forces caused by the belly pod opening.

Figure 12 shows the spectrograms of the vibrations measured at the pitch level plate – the base of the spectrometer – and the acoustic pressure in the belly pod. The sampling rate of the sensors is 1 kHz. In addition to Fig. 11c these plots depict the spectral contributions for different gimbal yaw angles. The microphone as well as the accelerometer clearly show the strongest vibration amplitudes between 200 and 250 Hz throughout the whole flight. This excitation probably originates aerodynamically at the belly pod side opening and appears at all gimbal yaw angles with varying amplitude. The highest value occurs at a gimbal yaw angle of 90° . Analysis of different sensor data indicates that the aerodynamic excitation changes with the position of the gimbal's shield and its opening relative to the airflow.

In order to avoid the strong vibrations at 90° , the CM measurements have been performed at a yaw angle of $86\text{--}87^\circ$. Another clearly visible frequency band at 40–60 Hz

in the accelerometer data is related to Eigen-frequencies of the gimbal and its shield. All other visible bands in both plots have considerable smaller amplitude.

3.2 Mechanical performance

A key indicator of the spectrometer function quality is the measurement of the moving mirror velocity using the reference laser system (Learner et al., 1996; Kimmig, 2001). It gives a good insight on the stiffness and damping characteristics of the system and on the impact of the vibrations on the IR measurement. The velocity measured with the reference laser system is directly proportional to the velocity of the moving cube corner and should ideally be constant. Unwanted velocity fluctuations are generated by fluctuations in the cube corner movement, but also by relative movements of the optical parts to each other. Such relative movements can e.g. be caused by independent vibrations of the optical components and the structure.

In order to quantify the stiffness of the spectrometer the inflight vibrations and the effects to the reference signal velocity are shown in Figs. 13 and 14. These pictures show the vibrations on the pitch level plate and the velocity signal recorded by the reference laser system over one interferogram at 15:18 UTC during flight 8 while the spectrometer is operated in CM. The environmental conditions remained stable at that time as shown in Fig. 11 in the previous section. The gimbal yaw angle varied only slightly in the range of 86.0 to 86.6° and the altitude was close to 15 km.

Figure 13 shows the power spectral density (PSD) of the vibration magnitude in the direction of scanner motion measured on the pitch level plate. This plot is derived from a single spectrum. Furthermore the integrated value of the PSD, the Cumulative Spectral Power (CSP), is displayed. The peaks in the PSD and steps in the CSP curve show very clearly that the largest power contribution of the vibrations is found around 50 and 220 Hz. The red curve shows the PSD limit for single vibration peaks in operational conditions as described in Sect. 2.1 and summarized in Table 1. It can be noted that the excitation at 217 Hz exceeds this requirement. The integral of the vibration excitation

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures



Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



the so called root mean square acceleration G_{rms} can be derived out of the CSP to 0.87 m s^{-2} . This is lower than the requirement which was set to 4 m s^{-2} .

The velocity signal detected by the reference laser for the same time interval shows a standard deviation of 9%, exceeding the target value of 5% given in Table 1. Figure 14 shows the amplitude spectrum of the velocity. The largest contribution to the velocity fluctuations is again around 217 Hz in accordance with the measured vibrations. Furthermore there are sharp peaks which are found at multiples of 6.41 Hz. They are caused by the thread of the scanner which has a rotation rate of 6.41 Hz. The second largest peak which can be found at 128.2 Hz is generated by the cogging torque of the twenty electrical coils in the stator of the drive.

The main part of the velocity fluctuations in the interferometer is caused by forced vibrations of the optical components induced by the strong acoustic and mechanical excitations around 220 Hz as shown in Fig. 12.

3.3 Thermal performance

The temperatures measured inside and outside the spectrometer and the flight altitude are shown in Fig. 15 for flight 19 on the HALO aircraft. The same flight was already discussed in Sect. 3.1. In Fig. 15b the temperatures inside the optic module and the coolant tank are shown. The sensor on the coolant tank, identified as “coolant tank” is the farthest away from the central body and shows the lowest temperature of the cooling system. The sensor “coolant tank at central body” is located at the interface between the central body and the coolant tank. It shows the highest temperature of the cooling system and the lowest temperature of the central block as expected. A third sensor, positioned in the IR-objective, shows the highest measured temperatures inside the optic module with the exception of the entrance and detector windows. The temperature of the latter is influenced by the environmental pressure, window heating and indirect heating by electronic components. Both window temperatures are shown in Fig. 15d.

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures



Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



ice capacity ensures a hold time of 24 h. Inflight measurements show that the thermal requirements could be fulfilled as the temperature drift could be kept below 1 K h^{-1} for most of the optical components at an operating temperature below 220 K.

The use of liquid CO_2 , a non-cryogenic product commercially and widely available in gas cylinders with unlimited holding time makes the instrument suitable for campaigns all over the world. Filling the coolant at ambient temperature allows using an uninsulated hose and therefore easy access through small hatches. It is conceivable to use the developed cooling techniques for any type of instrument, devices or calibration sources with similar thermal specifications.

Since the first scientific campaigns with GLORIA were successful, GLORIA will further participate in dedicated campaigns in 2016 and follow-on years (Riese et al., 2014). For future flights, the vibration analysis presented in Sect. 3.2 has resulted in modification plans for the HALO belly pod in order to reduce the aerodynamical excitation. The central body and the housing of the scanner will be replaced by a monolithic housing to increase the stiffness of the optic module and to reduce the influences of the vibrations. Further improvements concentrate on the thermal fluctuations of the entrance and the detector window. Different strategies are currently investigated: the mechanical interfaces at the detector side will be adapted for improving the insulation to VIP and stabilizing the temperature of the entrance window by controlled heating during measurements to reduce the temperature drift.

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AMTD

7, 10965–11010, 2014

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



References

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AMTD

7, 10965–11010, 2014

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



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AMTD

7, 10965–11010, 2014

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



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AMTD

7, 10965–11010, 2014

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



Table 1. Mechanical and thermal requirements for the GLORIA spectrometer.

Interferometer	Optical path difference	± 9 cm
	Velocity variation	$< 5\%$ RMS
Detector module integration	Thermal Input	Shall be minimized
	Integration concept	Modular/Demountable
Optics	Lens clear aperture	36.1 mm
	Maximum FOV	$4.1^\circ \times 4.1^\circ$
	Environment	Dry and clean
	Temperature	< 220 K
Cooling system	Temperature drift	< 2 K h $^{-1}$
	Agility	Yaw -20 to 140° Pitch -98 to 14° Roll -5 to 5°
Environmental conditions	Ambient temperature	200 to 320 K
	Ambient pressure	70–1000 hPa
	vibrations limits PSD during measurements	10 % of DO160C curve C
	vibrations limits G_{rms} during measurements	4 m s $^{-2}$ (0.412 g)
	Max. humidity	100 % r. H.
	Housing surface	Dry
Static loads for strength	Forward HALO	2 g
	Forward Geophysica	9 g
	Up-/downward	3.2 g/6.3 g
	Side-/backward	3 g/1.5 g

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



**The mechanical and
thermal setup of the
GLORIA
spectrometer**

C. Piesch et al.

Table 2. Operational characteristics of the GLORIA cooling system.

Cooling system	Volume coolant tank	2.2 L
	Operating temperature (in flight)	200 ... 220 K
Controlled cooling (liquid)	Liquid CO ₂ Consumption	24 kg d ⁻¹
	Liquid N ₂ Consumption	9 kg d ⁻¹
	Liquid N ₂ Cooling to 220 K	3.5 kg
Charging dry ice	Filling time	30 min
	Fill quantity	2.3 kg
Dry ice cooling	Temperature drift (on ground)	1.6 K h ⁻¹
	Temperature drift (in flight)	1 K h ⁻¹
	Stand-alone time	up to 24 h

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Table 3. Heat load on the cooling system.

Insulation	In laboratory	17 W
Window	Heating included	3 W
Internal heat dissipaters	Scanner drive with encoder, focus drive of IR-imaging optic and reference laser system	5 W
Detector	With compressor	5 W

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures



Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



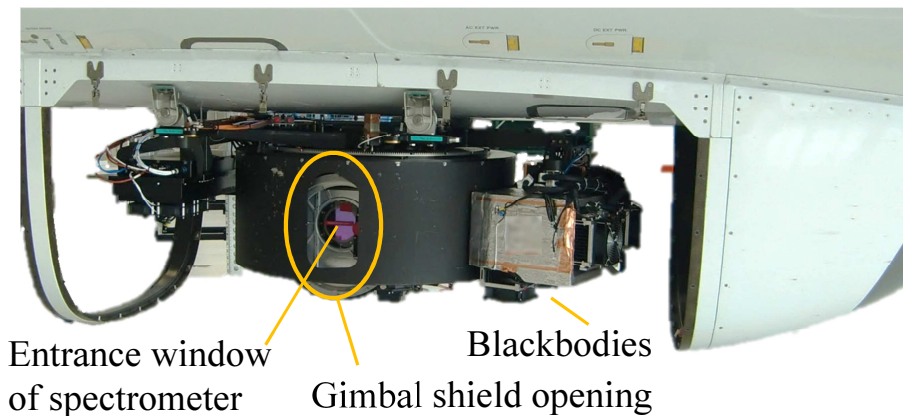


Figure 1. HALO with belly pod and opening for GLORIA (top) and belly pod with fairing dismounted showing the GLORIA instrument (bottom). The entrance window of the spectrometer is behind the opening in the gimbal's shield.

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion

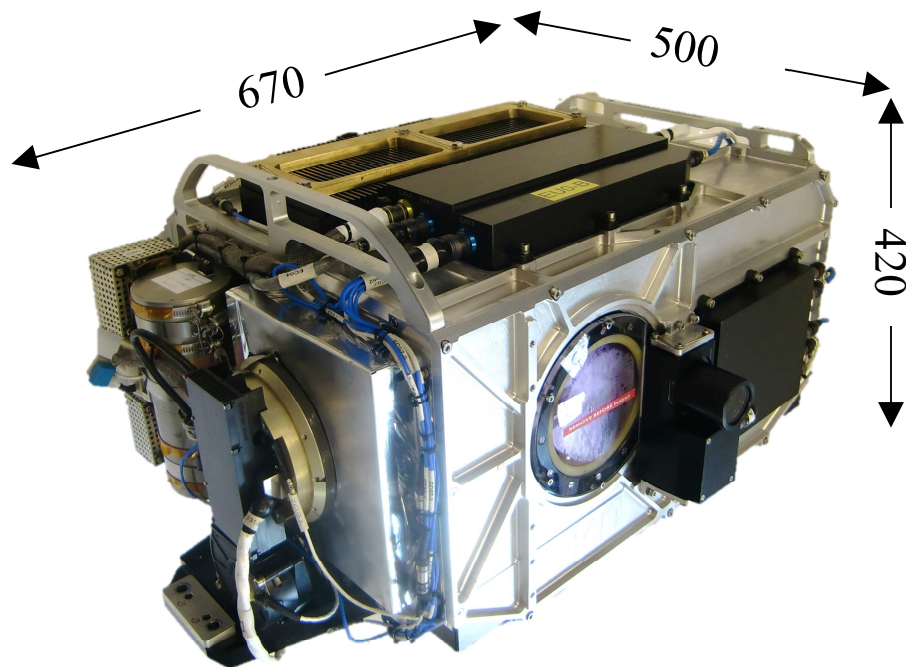


Figure 2. GLORIA Spectrometer ready for installation in the gimbal. Outer shell with electronic boxes (black), the entrance window (with cover), and the detector module (left side). The interferometer optics with cooling system is housed inside.

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion

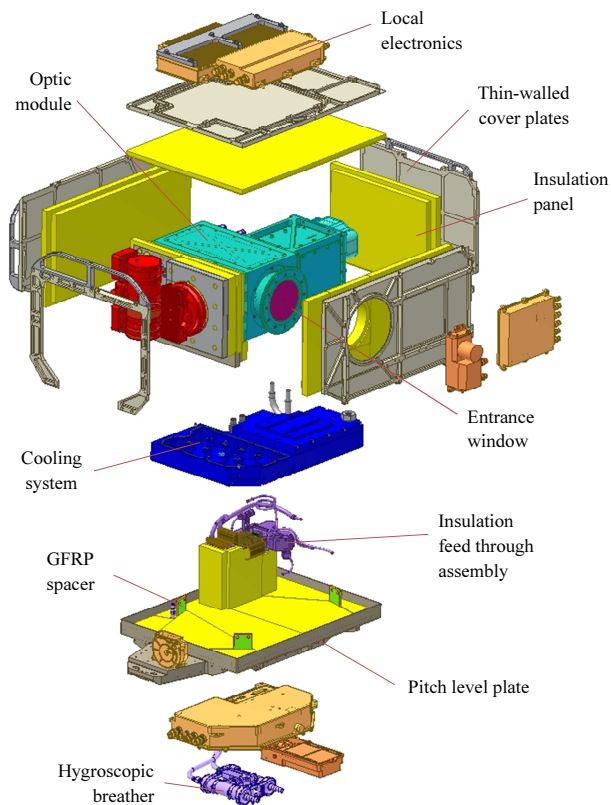


Figure 3. Exploded view of the GLORIA spectrometer showing components of the housing: pitch level plate, cover plates (grey), insulation (yellow), and local electronics (orange). Also shown is the optic module (cyan) with detector (red) and the cooling system (dark blue). External cables between electronic modules are not shown.

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

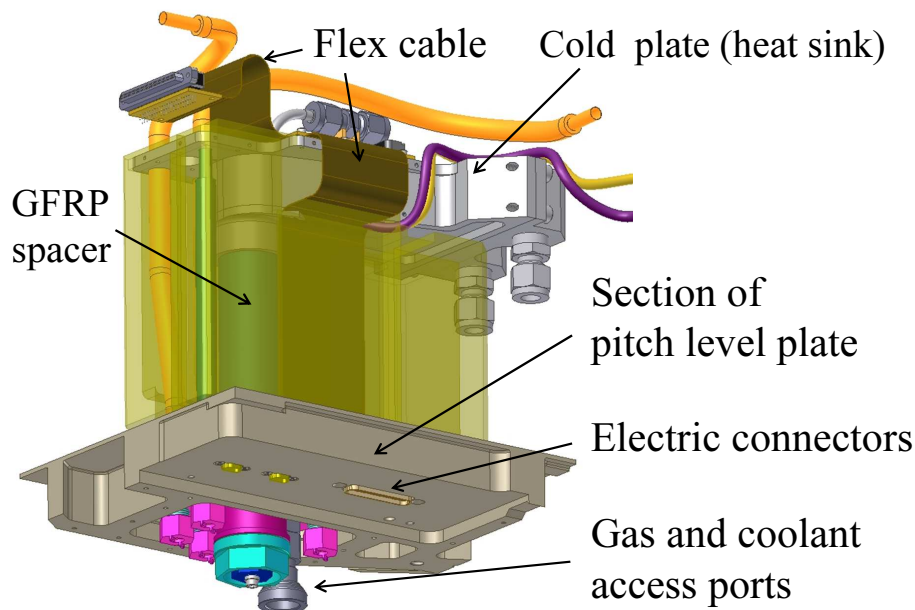


Figure 4. The insulation feedthrough assembly mounted on the pitch level plate (partly shown in gray). The heat input by tubes and cables between bottom part (ambient temperature) and top part (optic working temperature) is minimized.

[Title Page](#)[Abstract](#)[Introduction](#)[Conclusions](#)[References](#)[Tables](#)[Figures](#)[◀](#)[▶](#)[◀](#)[▶](#)[Back](#)[Close](#)[Full Screen / Esc](#)[Printer-friendly Version](#)[Interactive Discussion](#)

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

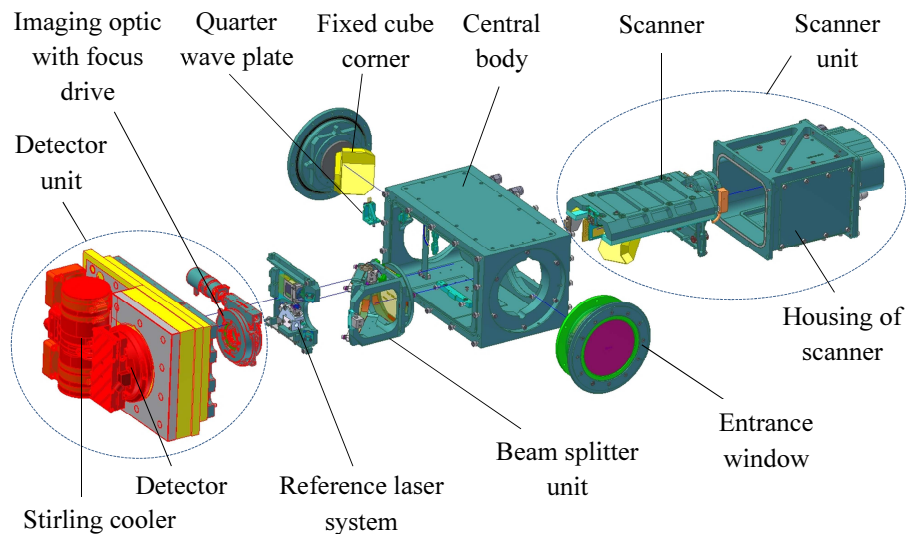


Figure 5. Exploded view of the optic module illustrating the main opto-mechanical (cyan) and optical components of the interferometer, the entrance window, the imaging optic and the detector system (red).

[Title Page](#)[Abstract](#)[Introduction](#)[Conclusions](#)[References](#)[Tables](#)[Figures](#)[◀](#)[▶](#)[◀](#)[▶](#)[Back](#)[Close](#)[Full Screen / Esc](#)[Printer-friendly Version](#)[Interactive Discussion](#)

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

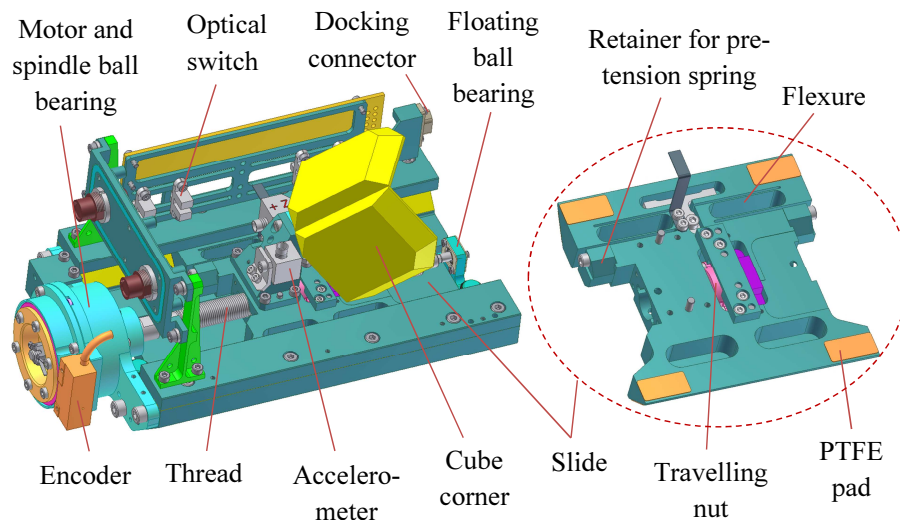


Figure 6. Complete scanner with slide (left) and detail of slide (right). The preloaded dovetail guidance with high contact area and a thread for the feed motion was chose to get a stiff and rigid design.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

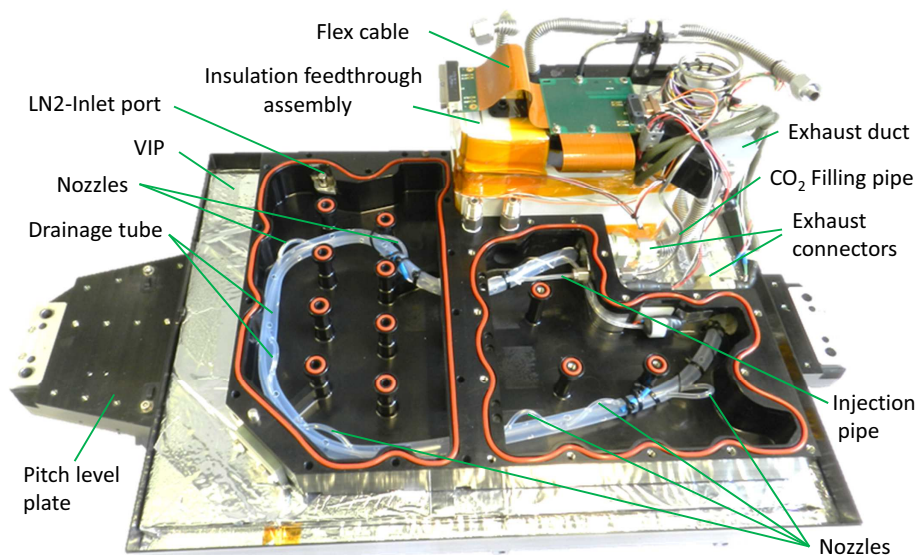


Figure 7. Coolant tank internal view with insulation feedthrough assembly (top right) mounted on the pitch level plate. The injection pipe is used to spray liquid CO_2 into the coolant tank through small nozzles. The liquid CO_2 is introduced over the feedthrough assembly into the tank. Precooling with LN_2 is possible over the dedicated LN_2 inlet. The exhaust gas is guided out by the drainage tube and heat sinking parts of the feedthrough.

[Title Page](#)[Abstract](#)[Introduction](#)[Conclusions](#)[References](#)[Tables](#)[Figures](#)[◀](#)[▶](#)[◀](#)[▶](#)[Back](#)[Close](#)[Full Screen / Esc](#)[Printer-friendly Version](#)[Interactive Discussion](#)

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion

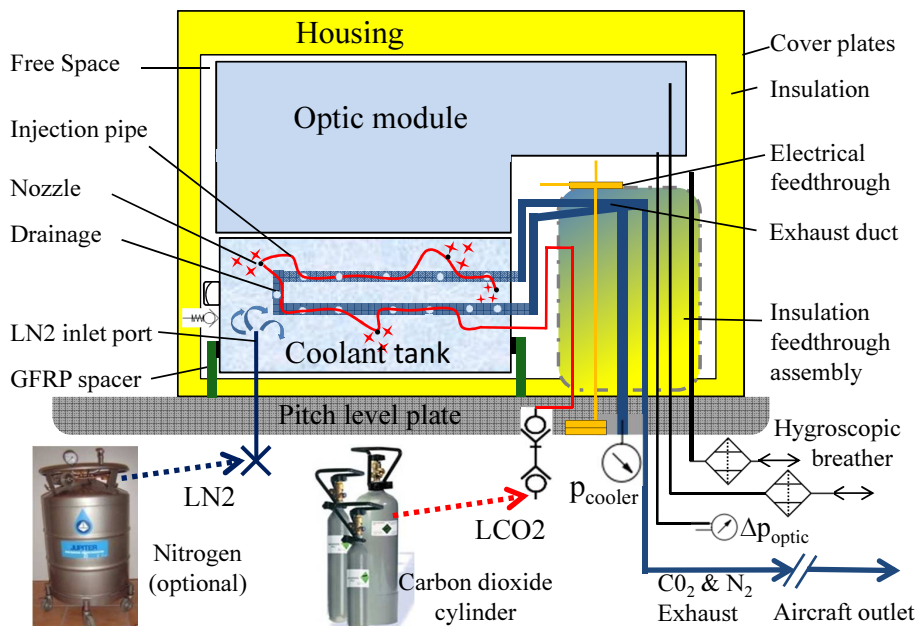


Figure 8. Functional thermal setup of the GLORIA spectrometer. Injection of liquid CO₂ allows charging dry ice through in situ production of CO₂ snow in the coolant tank. Alternatively controlled cooling is possible by injecting small amounts of liquid CO₂ or liquid N₂.

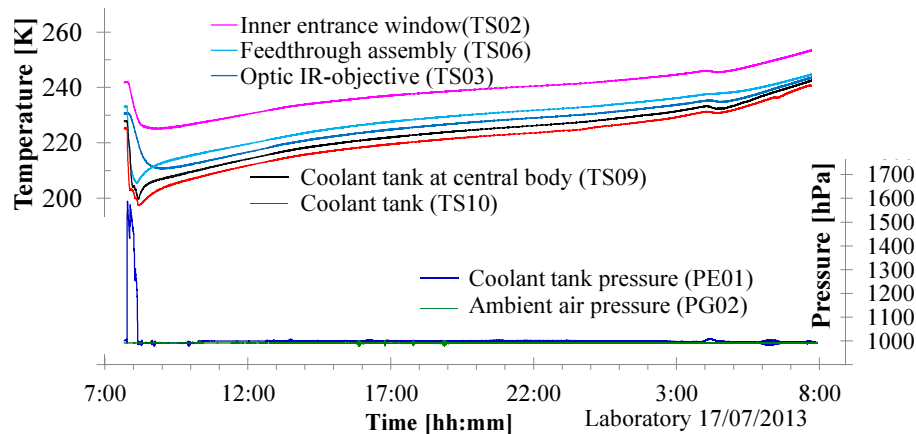


Figure 9. Spectrometer temperature and pressure values during laboratory testing of dry ice filling (about 30 min) followed by laboratory operation without external coolant input.

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

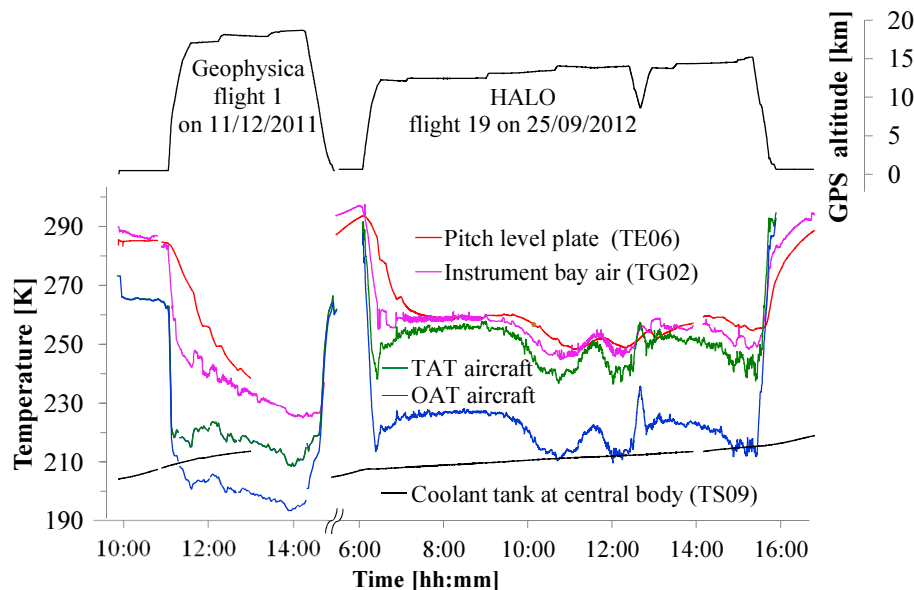


Figure 10. Ambient temperatures during typical operation of GLORIA on the Geophysica (left side) and HALO aircraft (right side). The evolution of ambient and instrument temperature is illustrated along flight altitude profile (OAT = temperature of undisturbed outside air, TAT = includes the hydrodynamic temperature rise caused by the airplane velocity).

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

◀

▶

◀

▶

Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion



The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures



Back

Close

Full Screen / Esc

Printer-friendly Version

Interactive Discussion

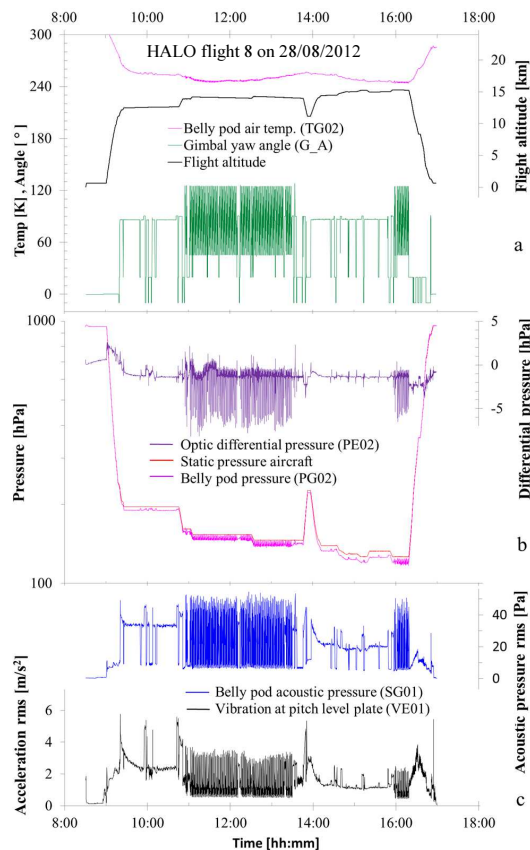


Figure 11. Ambient conditions of GLORIA during flight 8 on HALO on 28 August 2012: **(a)** belly pod air temperature (pink), flight altitude (black), and gimbal yaw angle (green), **(b)** optic differential pressure (purple), static pressure aircraft (red), and belly pod pressure (magenta), and **(c)** belly pod acoustic pressure (blue) and spectrometer vibrations (black).

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures

[Back](#)

Close

Full Screen / Esc

[Printer-friendly Version](#)

Interactive Discussion

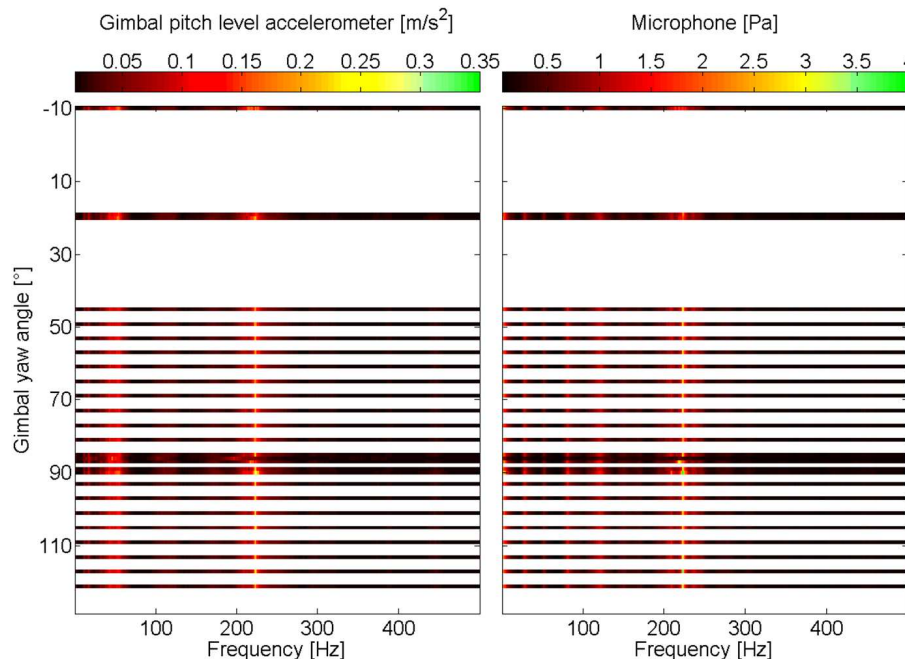


Figure 12. Spectrogram of vibration and acoustic pressure measurements of the pitch level plate for different gimbal yaw angles averaged over the whole flight 8 on HALO on 28 August 2012. Only discrete gimbal yaw angles are illustrated as only those angles are used during the measurements.

The mechanical and thermal setup of the GLORIA spectrometer

C. Piesch et al.

Title Page

Abstract

Introduction

Conclusions

References

Tables

Figures



[Back](#)

Close

Full Screen / Esc

[Printer-friendly Version](#)

Interactive Discussion

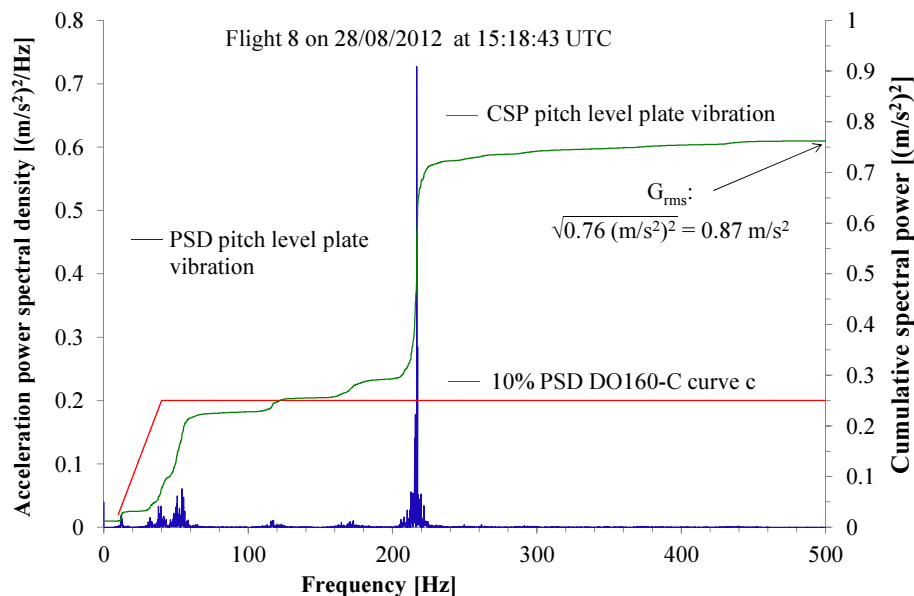


Figure 13. Vibration measurement at pitch level plate during flight 8 on HALO during chemistry mode measurement at gimbal yaw angle of approximately 86° . Shown are the measured Power Spectral Density (PSD) (blue) and its Cumulated Spectral Power (CSP) (green) as well as the PSD of the assumed limit of 10 % DO-160C curve c (red) for single vibration peaks.

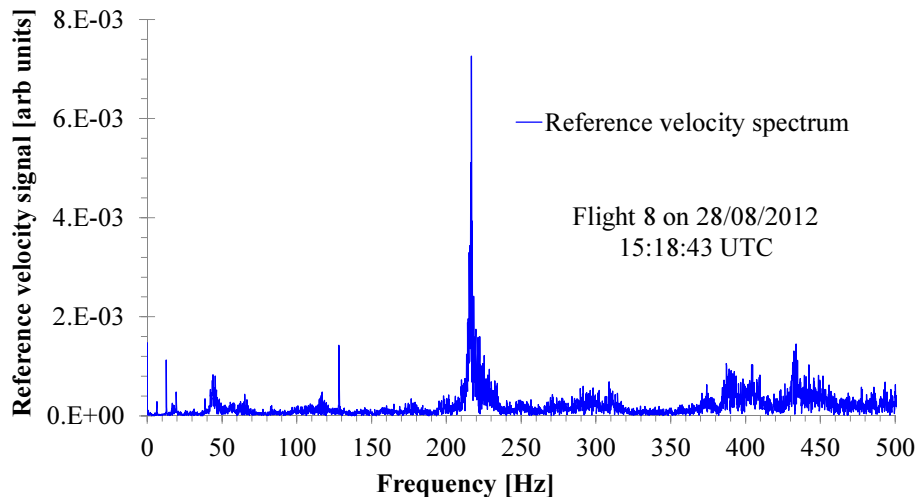


Figure 14. Spectrum of the velocity signal from the interferometer measured with the reference laser during the same time period which is illustrated in Fig. 13. The largest peak is caused by the strongest vibration identified in Fig. 13.

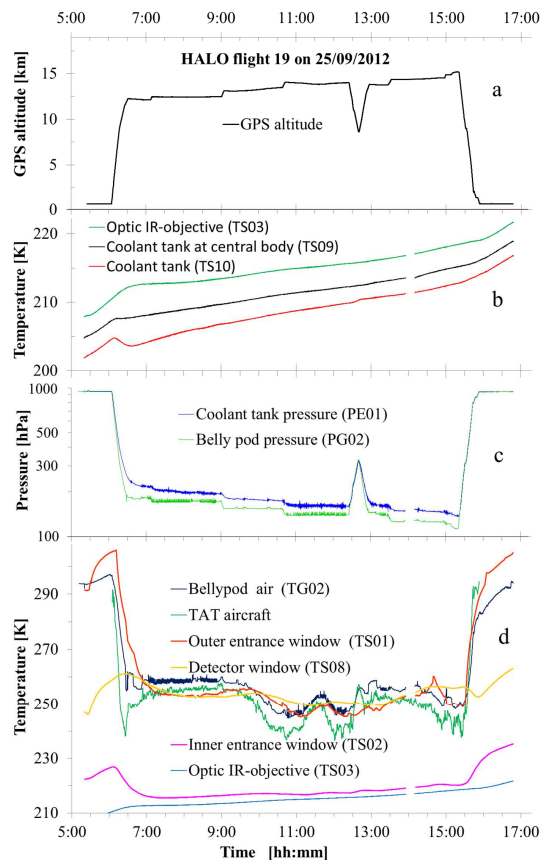


Figure 15. Conditions observed during flight 19 on HALO. The plots show **(a)** flight altitude, **(b)** optic and coolant tank temperatures, **(c)** coolant tank pressure, **(d)** aircraft and window temperatures.